



Thales Cryogenics

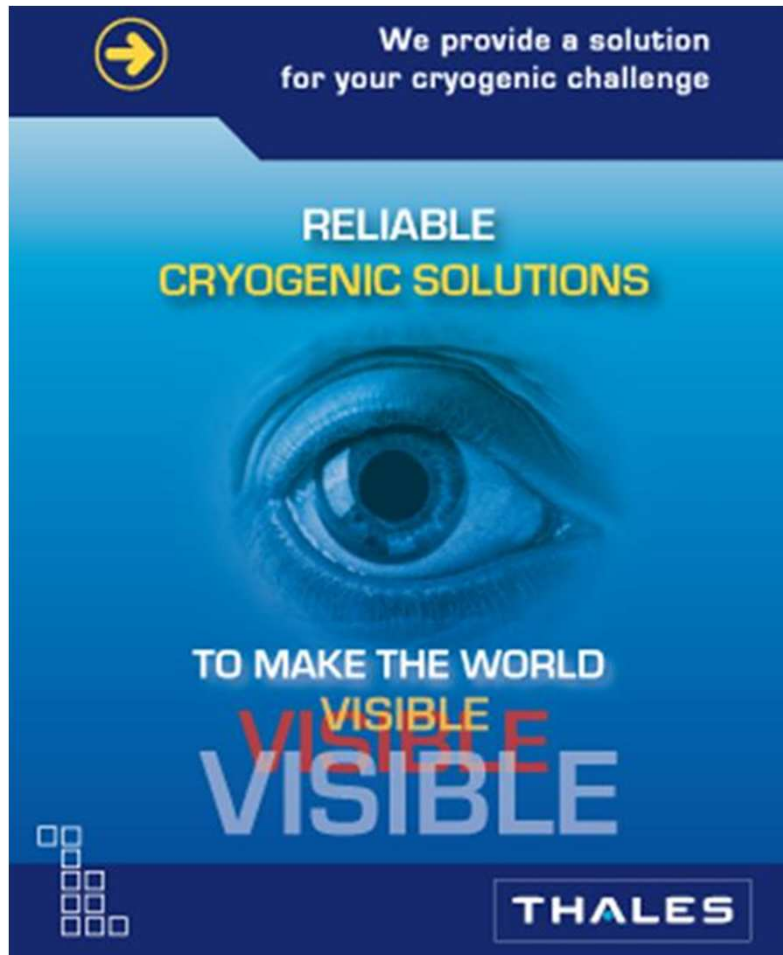
Compressor development for MTG pulse tube cryocooler

M. Meijers, J. Mullié, Th. Rijks,
F. van Wordragen, A. Göbel,
B. Schalkwijk, N. Sanchez-Ciudad,

5th European Space Cryogenics Workshop

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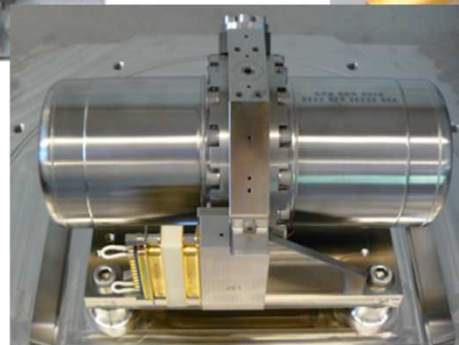
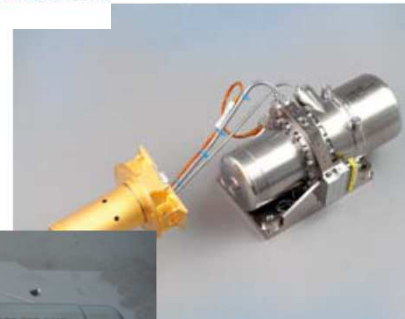
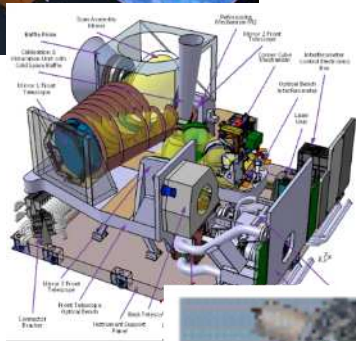
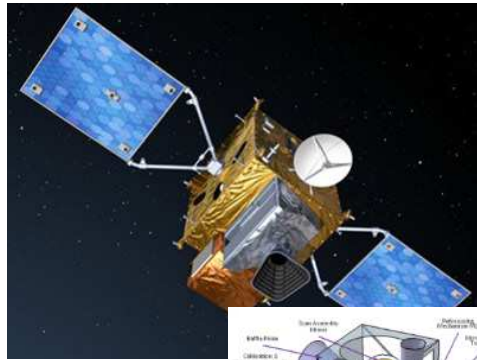
- **Introduction**
 - Thales Cryogenics
 - Large Pulse Tube Cooler Compressor history
- **LPTC compressor features & test results**
 - Efficiency
 - Lifetime
 - Self-induced vibrations
- **Summary & Questions**

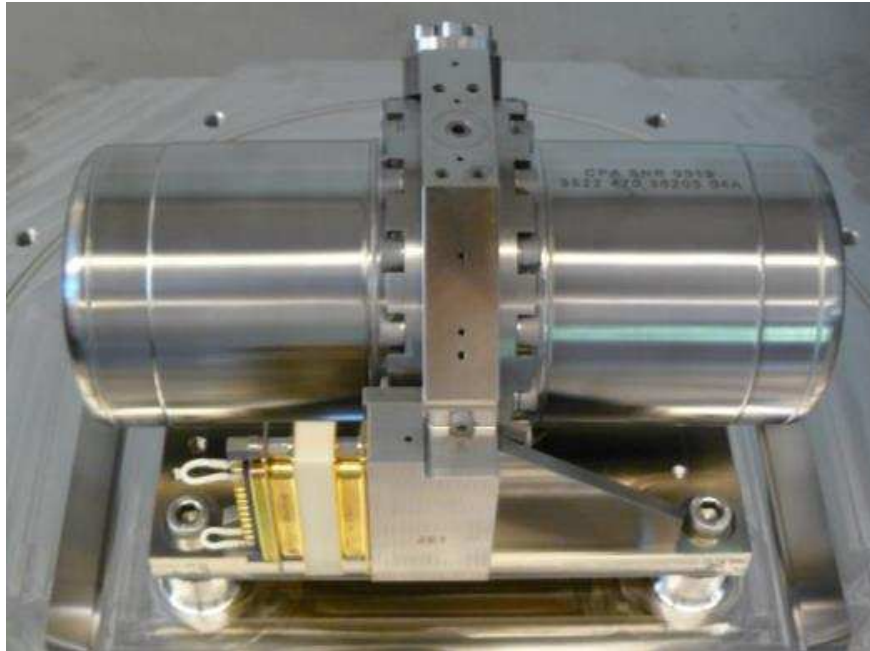


The advertisement features a dark blue background with a large, close-up image of a human eye in the center. The text is arranged as follows: at the top left, a yellow arrow icon points right, followed by the text 'We provide a solution for your cryogenic challenge'. Below this, the words 'RELIABLE CRYOGENIC SOLUTIONS' are written in white and yellow. The eye image is the central focus. Below the eye, the text 'TO MAKE THE WORLD VISIBLE' is written in white, with 'VISIBLE' repeated in large, bold, red letters. At the bottom left, there is a small graphic of a grid of squares. At the bottom right, the 'THALES' logo is displayed in white.

- Thales Cryogenics develops and manufactures cryogenic coolers for military, civil and space applications.
- Wide range of cryocooler products and cooler drive electronics:
 - Linear Stirling,
 - pulse tube and
 - Rotary coolers and electronics
- World leader in extremely high-reliable cryocoolers (life time > 100.000 hours demonstrated)
- Space flight heritage at cooler level in US defense satellites
- Delivery of required ground support equipment.

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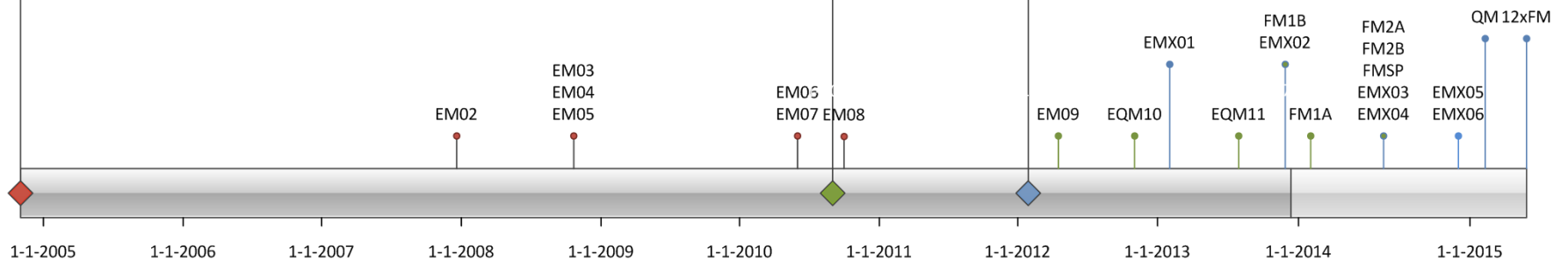
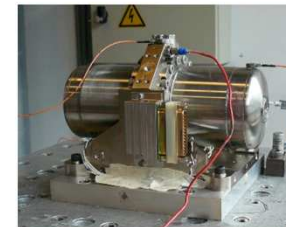
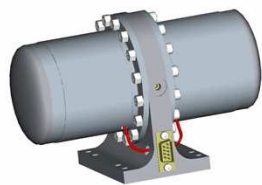
- Primary function PV power generation
 - Max input power 180 W
 - Resonance frequency 55 – 60 Hz
 - Mechanical power 120 W
 - Filling pressure 31 - 34 bar
 - Mass 4.9 kg
 - Max. swept volume 3.77 cc
-
- Dual opposed pistons
 - Moving magnet
 - Flexure suspended pistons
 - Dynamic pressure seal
 - Ti6Al4V pressure vessel
 - Dismountable motors on central bracket



1-11-2004
LPTC TRP

1-9-2010
Start CSO-MUSIS
Program

27-1-2012
Start MTG
Program

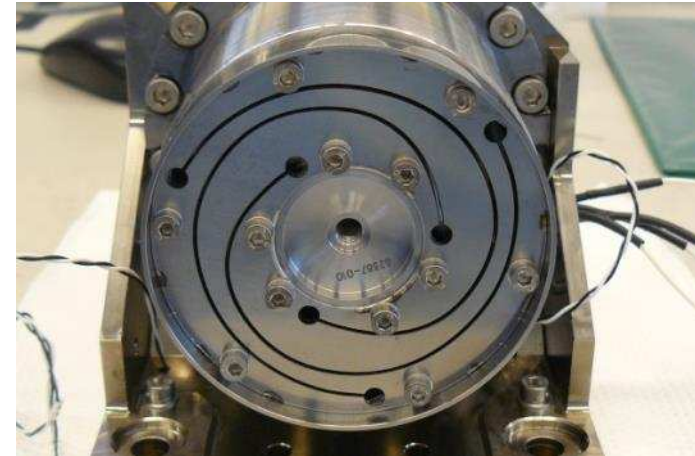


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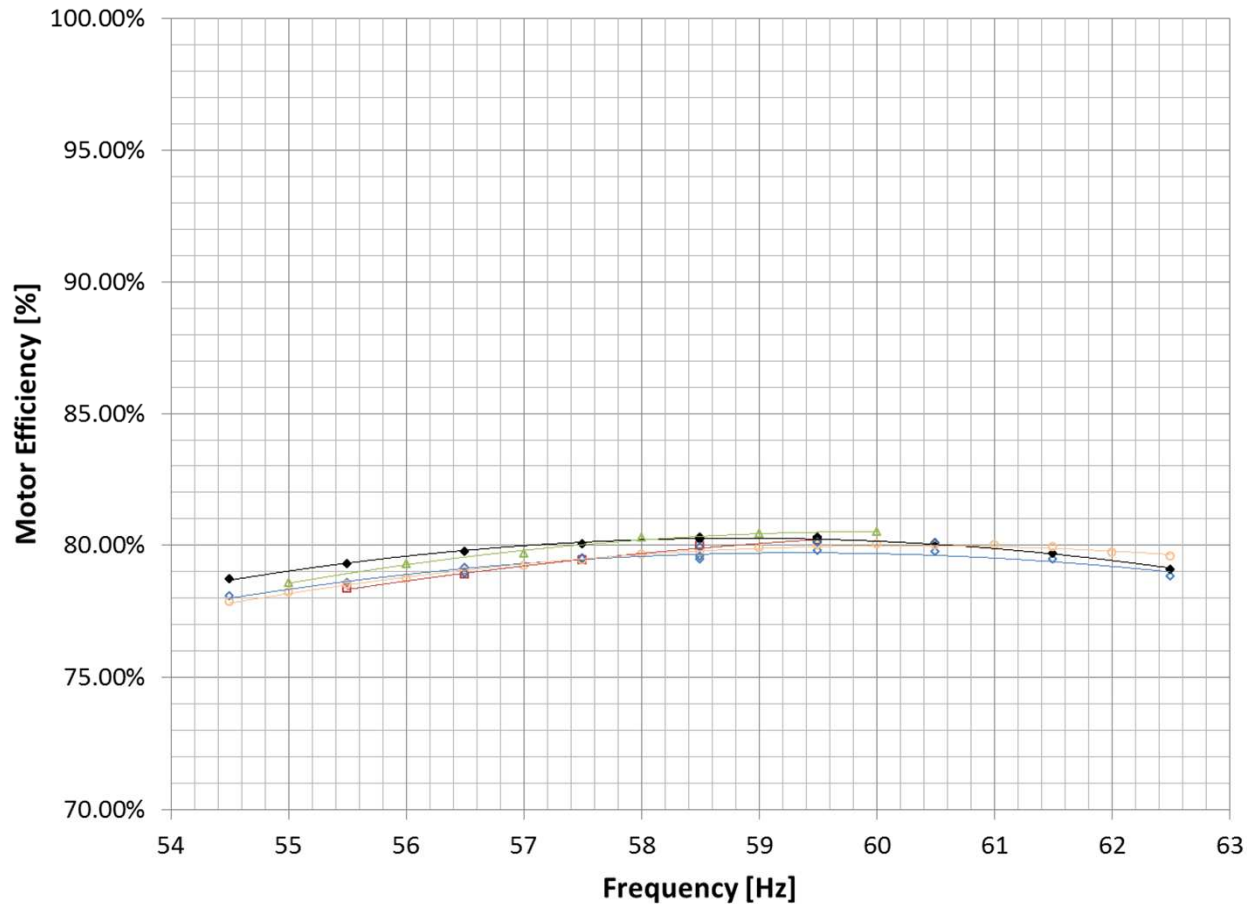


High Efficiency

- **Linear motor design**
- **High motor constant:**
 - NdFeB magnets
 - High coil filling factor
- **Minimizing eddy currents:**
 - Powder composite material stators
 - Ti6Al4V structural parts
- **Minimizing flow losses**
 - Very small piston gap providing dynamic pressure seal.
 - Piston with liner suspended in flexures
 - Minimizing contact forces
 - Minimizing friction forces

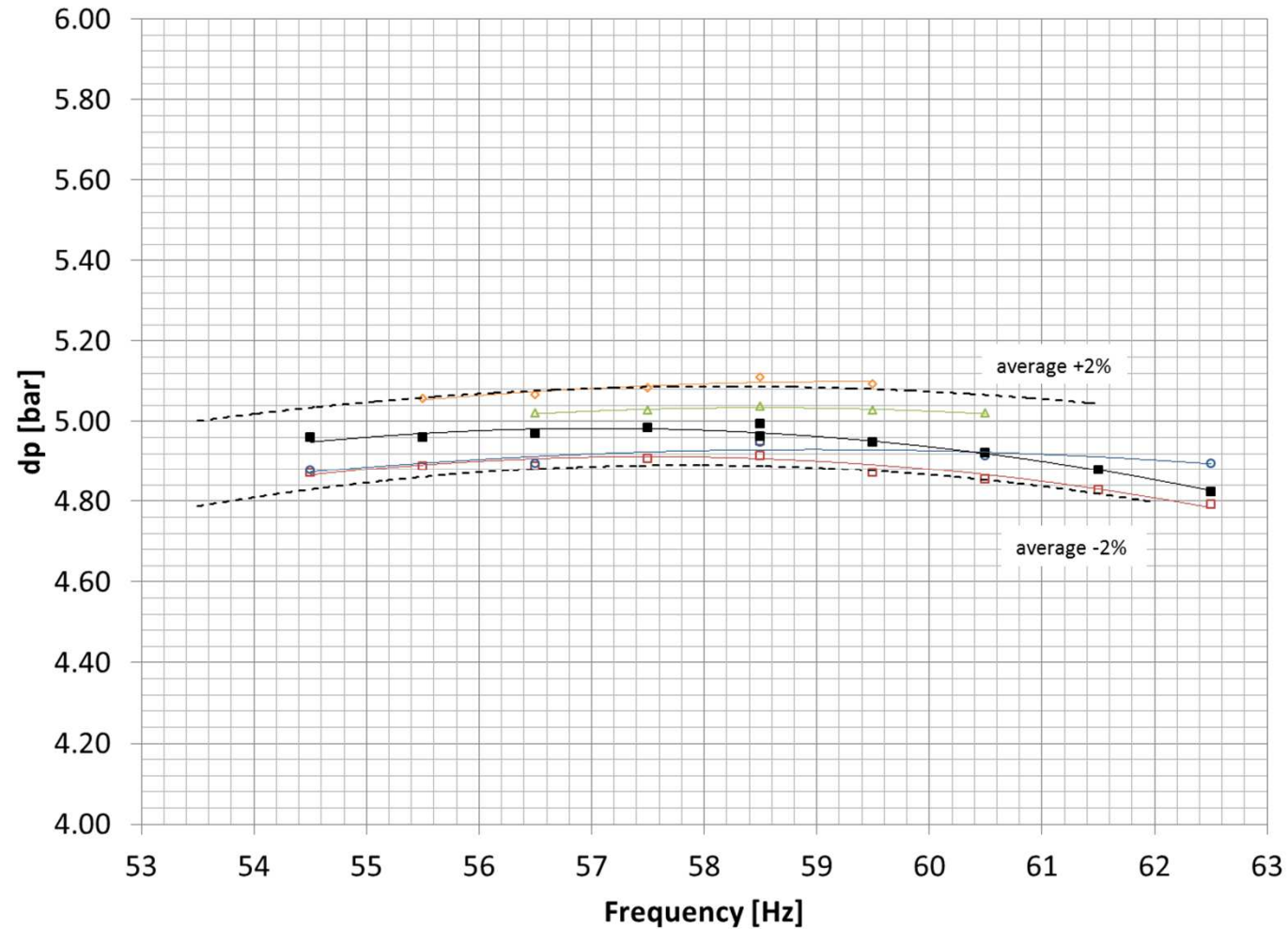


Measured motor efficiencies for delivered compressors



- Tested against a representative pulse tube delivering $\pm 2\text{W}$ at 50K
- Motor efficiency defined in this graph as $\eta = \frac{P_{in} - I^2R}{P_{in}}$
- Input power 160 W

Measured pressure waves for delivered compressors



- Tested against a representative pulse tube delivering $\pm 2W$ at 50K
- Input power 160 W

long lifetime

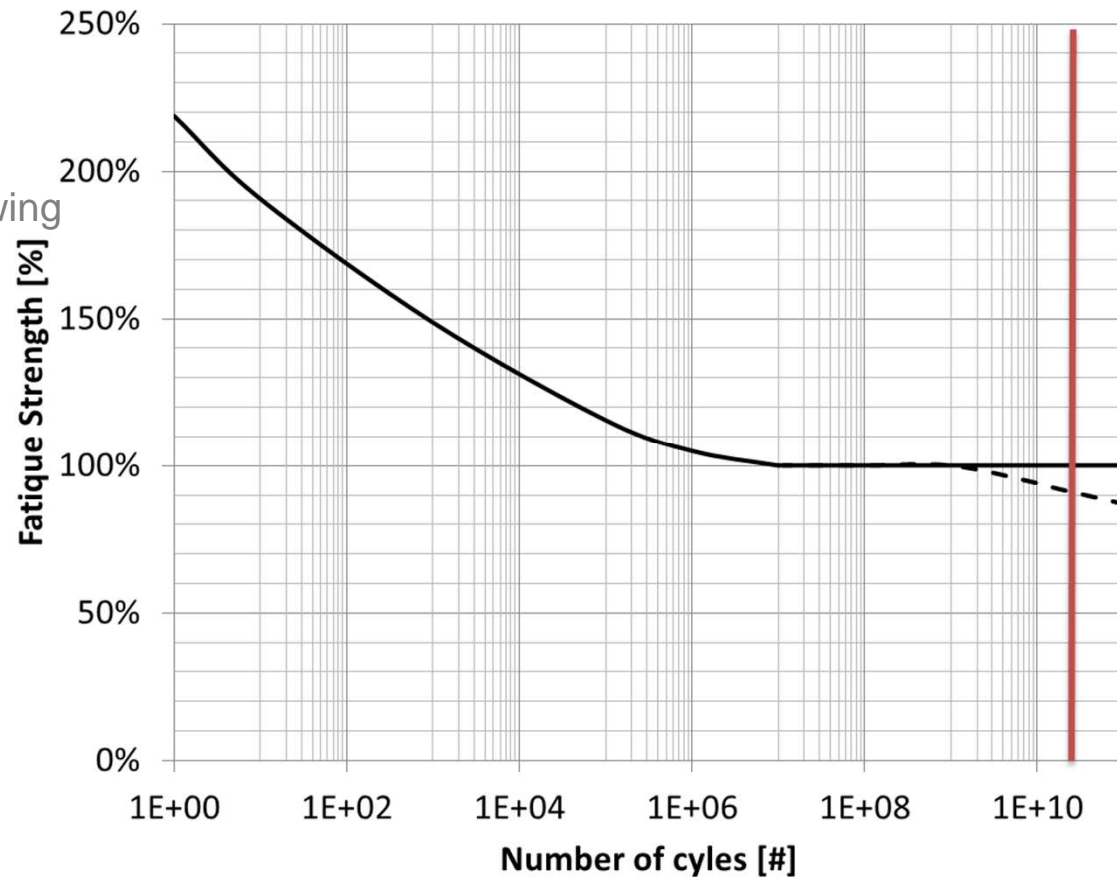
- **Leak tightness**
 - Minimising number of seals by EB-welding as much as possible
 - Using metallic C seals when unavoidably (requirement on motor accessibility)
- **Low off-gassing to He working gas**
 - Coils outside the pressurized part containing He working gas
 - Encapsulated stators
 - Effective cleaning processes, bake-outs, flushes, etc.



Long lifetime

- **Minimise risk on dynamic loaded structures**
 - No moving coil leads
 - Conservative flexure design:
 - Use of flexure material allowing high fatigue loads
 - High design margins
 - Photo-etching: minimizing manufacturing stresses
 - Thorough crack analyses of dynamically loaded welds

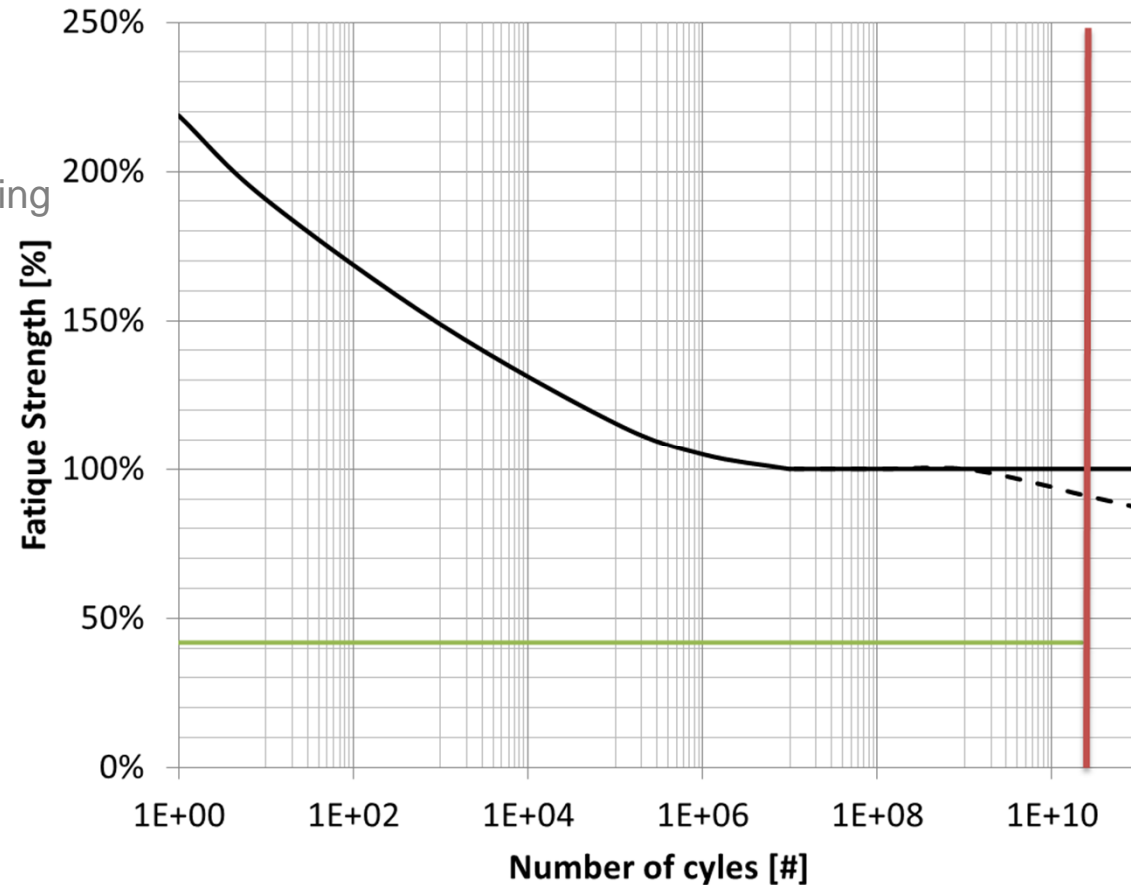
- Literature Material S/N curve
- - - Giga-cycle regime (C. Bathias)
- MTG Compressor operational lifetime



Long lifetime

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- Literature Material S/N curve
- - - Giga-cycle regime
- MTG Compressor operational lifetime
- Flexure stress at maximum input power

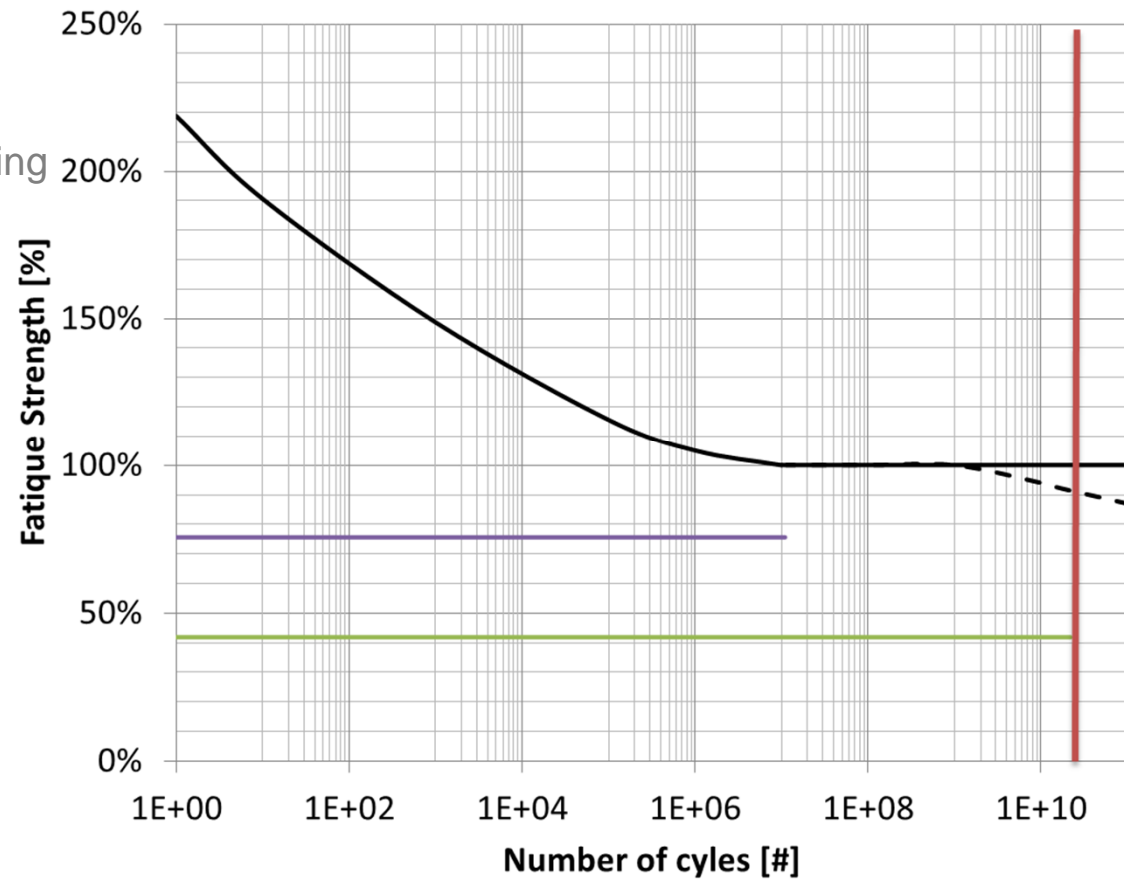


Long lifetime

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- Literature Material S/N curve
- - - Giga-cycle regime
- Flexure stress at maximum input power
- In assembly screening test level
- MTG Compressor operational lifetime



Low self-induced vibrations

- **Design**

- Dual opposed pistons

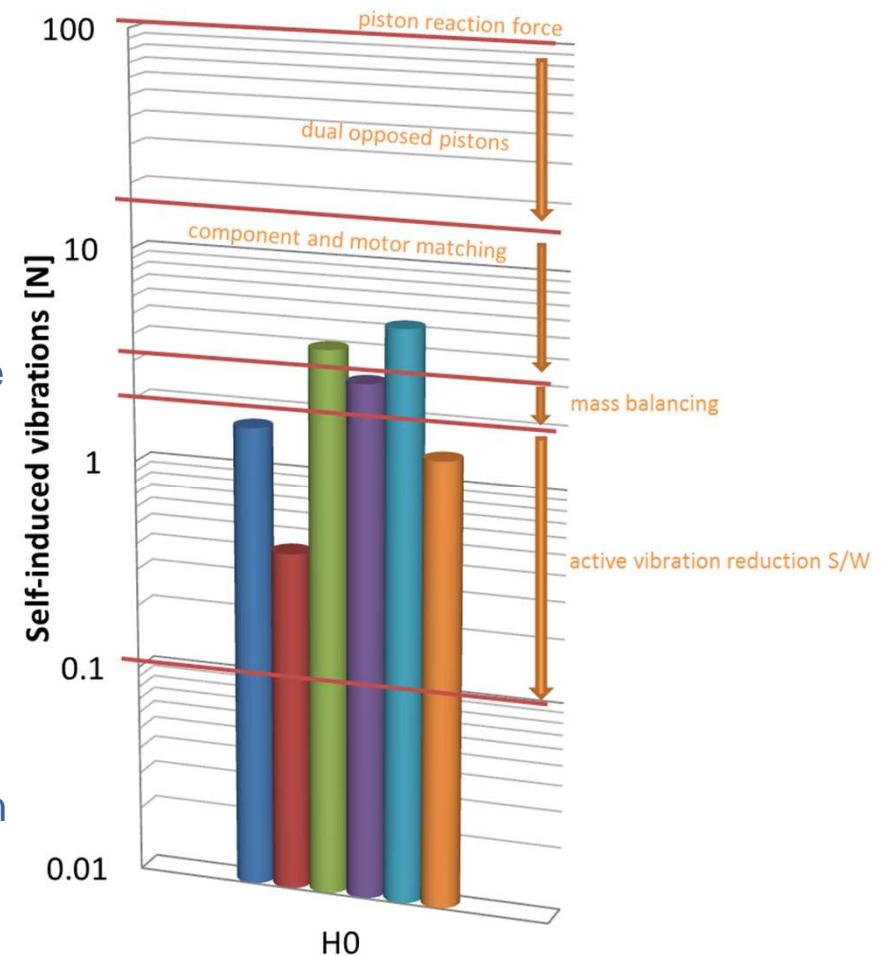
- **Assembly Strategy**

Minimize piston force amplitude and phase differences between opposing motors by

- matching of motor components
- matching of motors
- mass balancing of motors

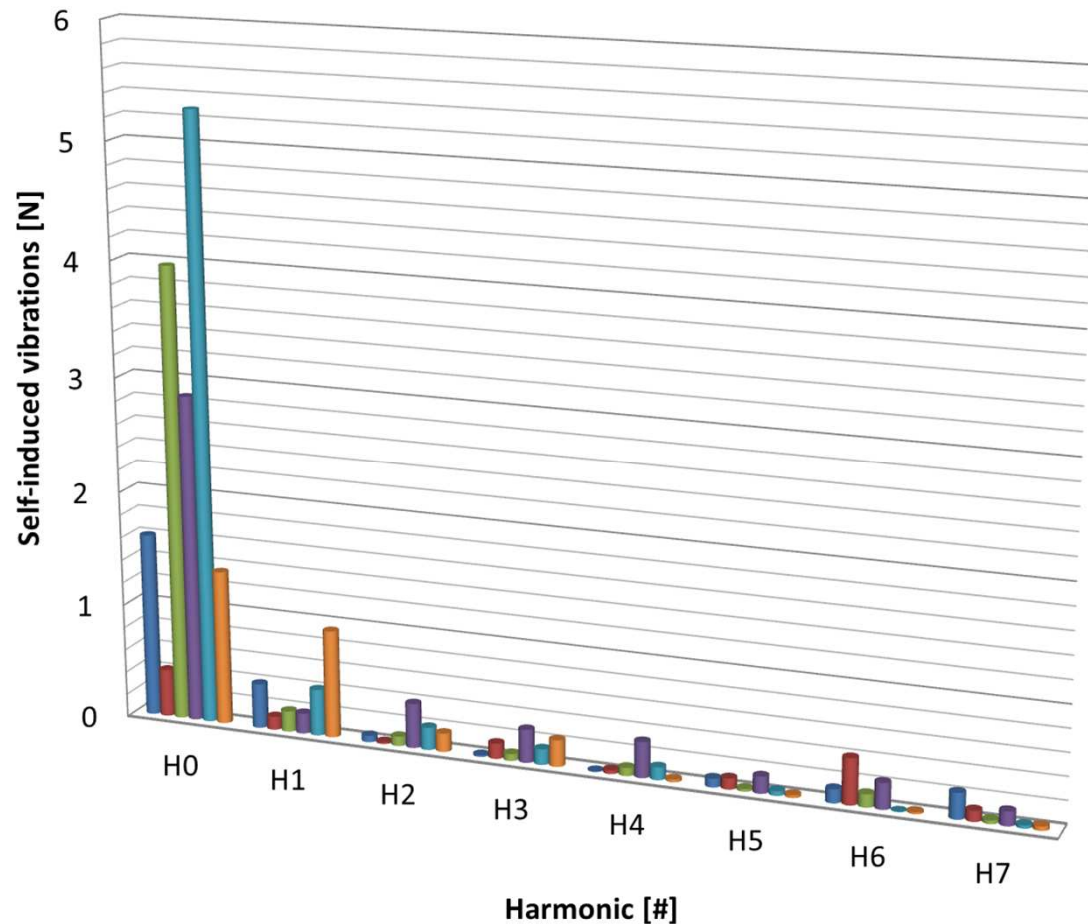
- **Potential further reduction**

- Application of active vibration reduction algorithms



Measured piston axis self-induced vibrations for delivered compressors

Piston axis self-induced vibrations

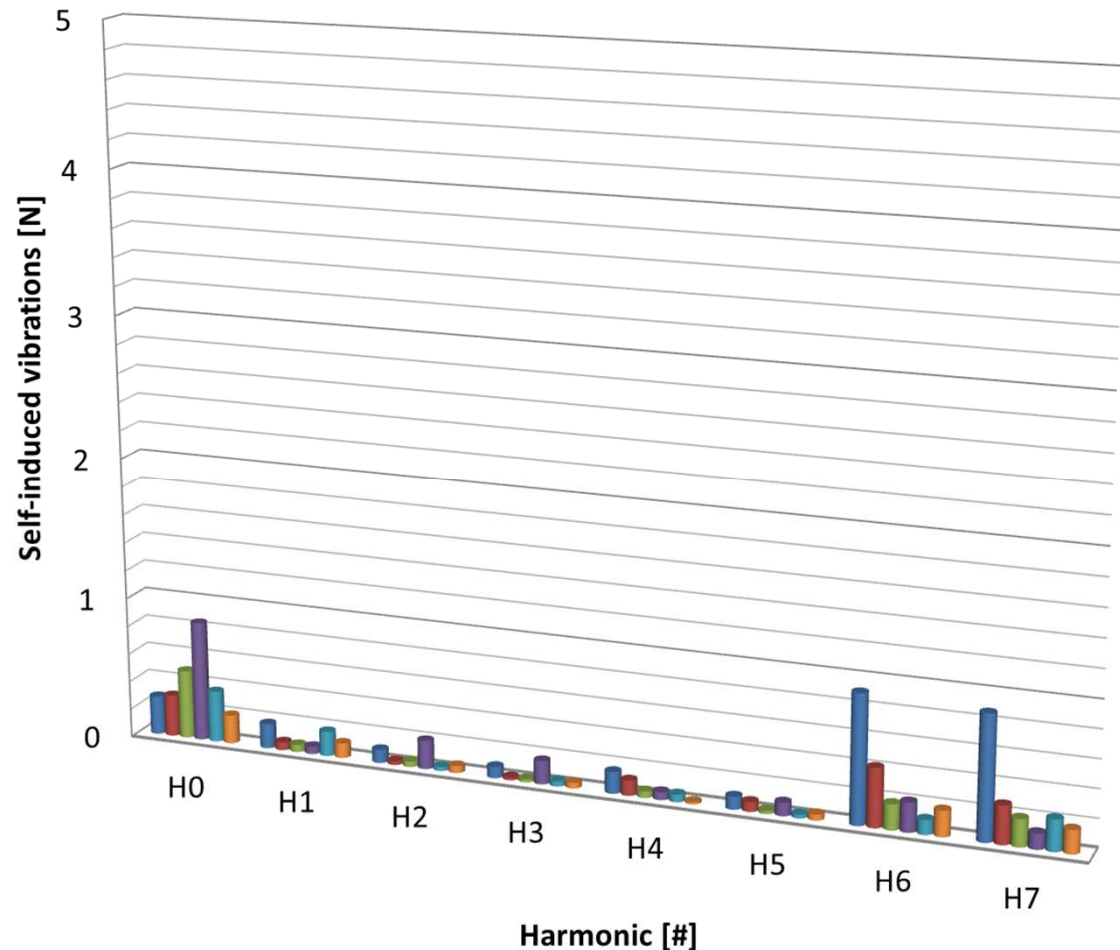


- Tested against a representative pulse tube delivering $\pm 2W$ at 50K
- Input power: 160 W
- Drive frequency: 58.5 Hz

Delivered compressors measurements

- Mass-balanced
- Different levels of motor matching

Transverse axis self-induced vibrations

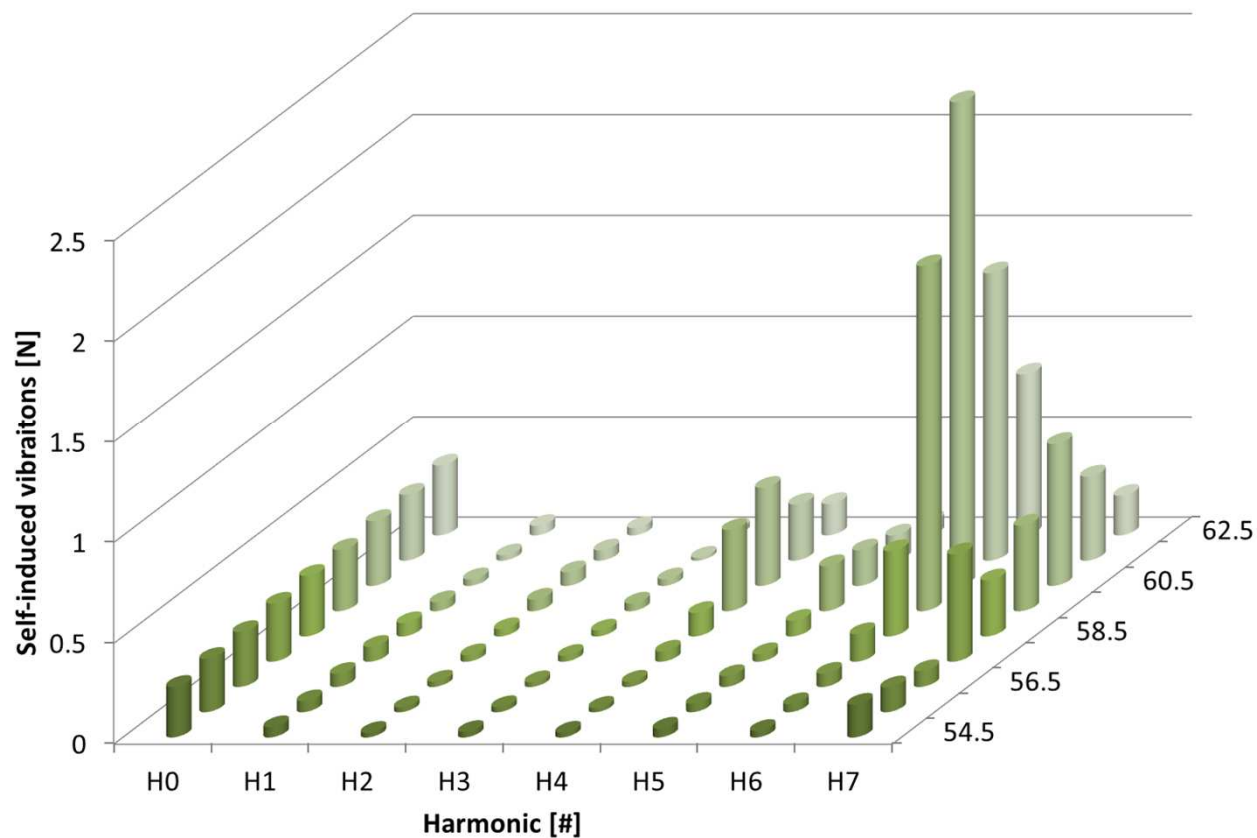


- Tested against a representative pulse tube delivering $\pm 2W$ at 50K
- Input power: 160 W
- Drive frequency: 58.5 Hz

Delivered compressors measurements

- No direct relation between piston and transverse axis is found

Transverse axis self-induced vibrations



- At certain drive frequencies, some harmonics coincide with structural resonance frequencies
- Confirmed by compressor vibration tests (shaker)
- Mapping provides tuning possibilities for customers

LPTC compressor

- **Is in a mature design status**
 - Currently building and delivering FM's for MUSIS-CSO project
 - First 2 EM's delivered for MTG, aiming for qualification mid 2014
- **Measured performance tests results as expected**
- **Lifetime aspects and potential risks are thoroughly reviewed in both projects**
- **Self-induced vibrations**
 - are limited by several passive methods
 - have the potential to be reduced to levels below 0.1 N in the piston axis with active vibration reduction (see presentation 7.1)



Piston axis self-induced vibrations vs. drive frequency

